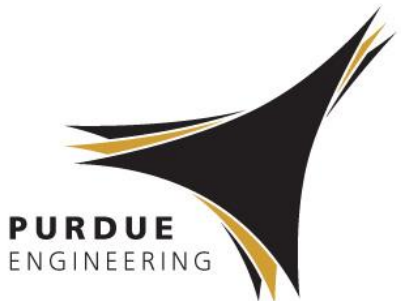


Lectures 10-12: Torsion members

Lecture Book: Chapter 8

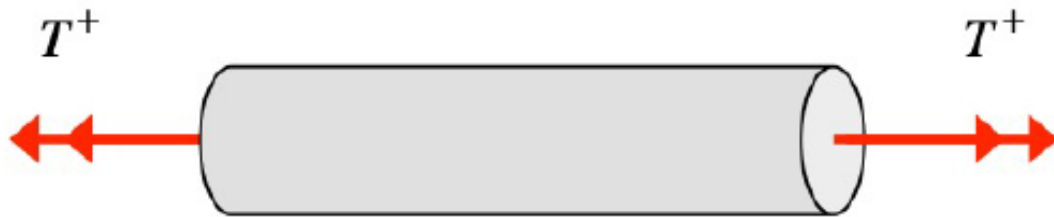
Joshua Pribe

Fall 2019



Objectives

- Derive important relations for pure torsion of a circular bar
 - Shear strain and angle of twist
 - Shear stress and shear strain
 - Shear stress and torque
 - Torque and angle of twist
- Compare with our equations for uniaxial loading
- Complete several examples on calculating shear stress

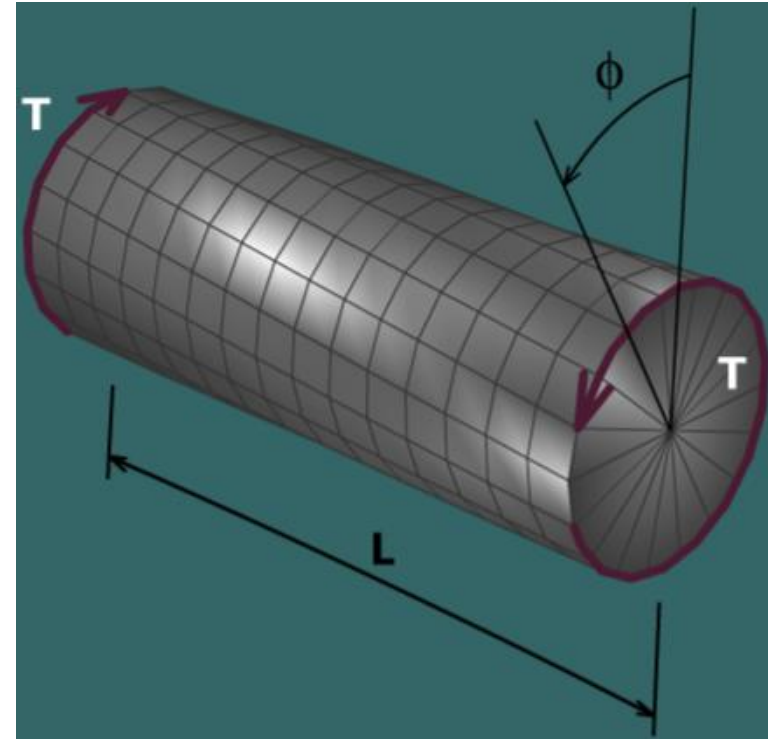


Torsion members: Assumptions

Lecture Book: Chapter 8, Page 3

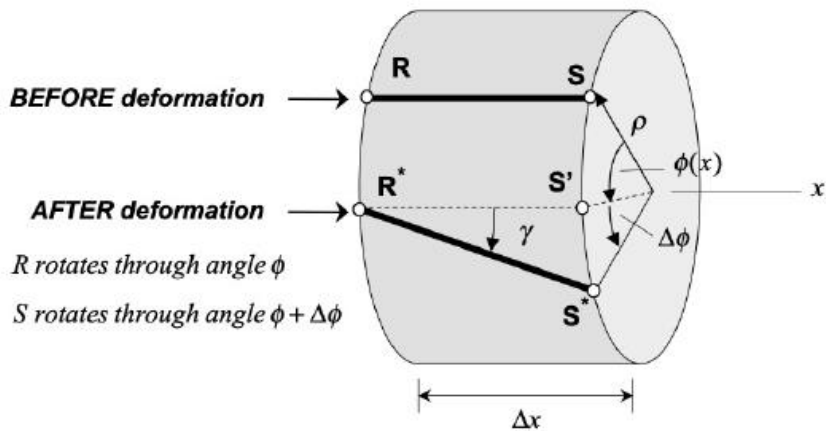
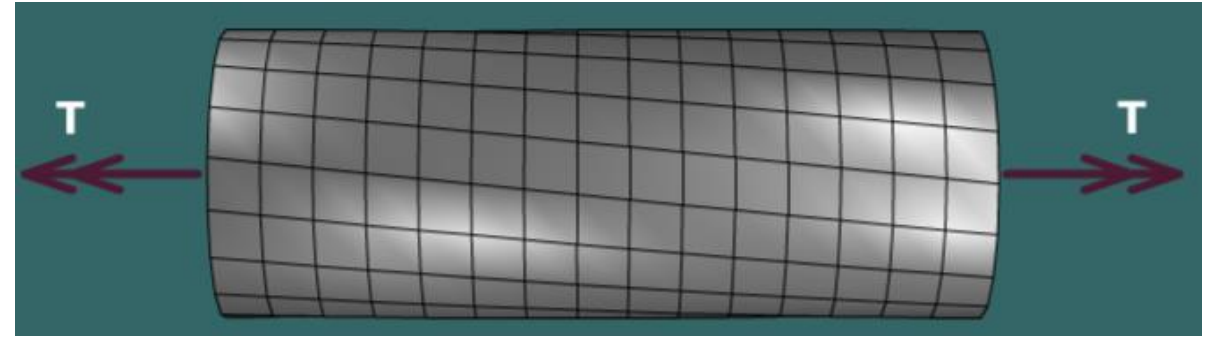
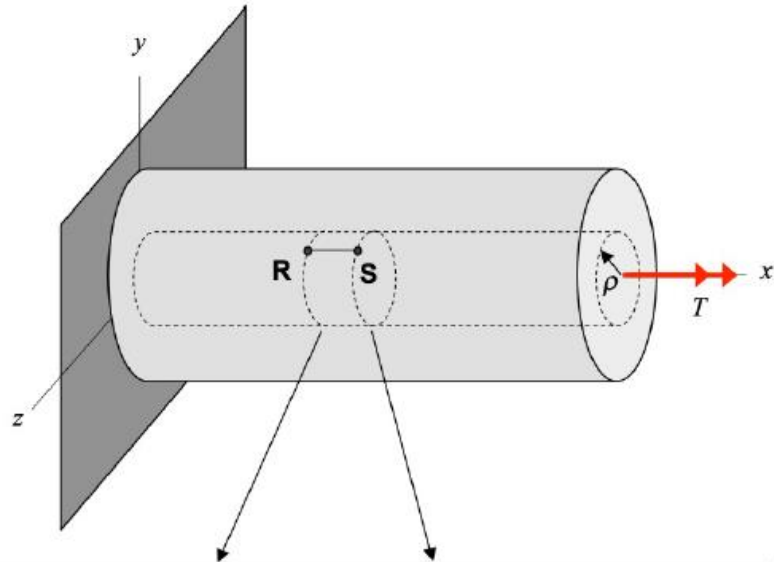
Assumptions:

- Only circular cross section shafts will be considered.
- Cross sections remain perpendicular to shaft axis after deformation (this is not a valid assumption, in general, for non-circular cross section bars).
- Radial lines remain radial (i.e. every point on a cross section rotates by the same amount).
- Shaft axis remains straight.



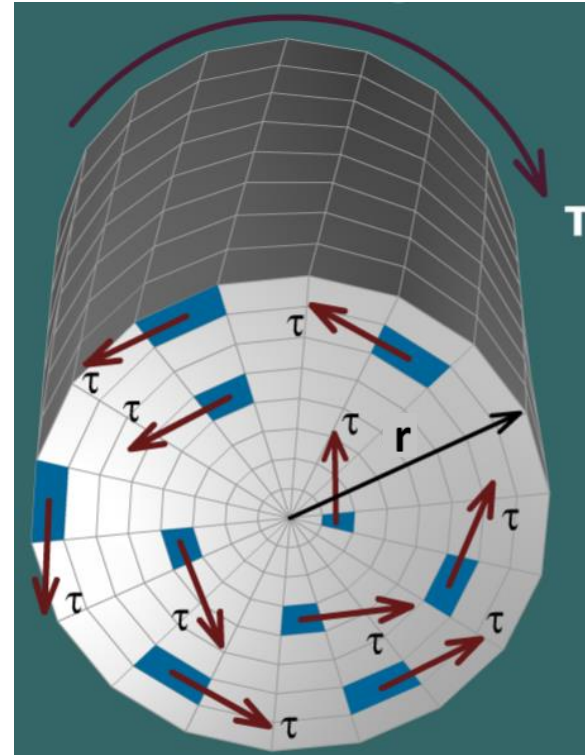
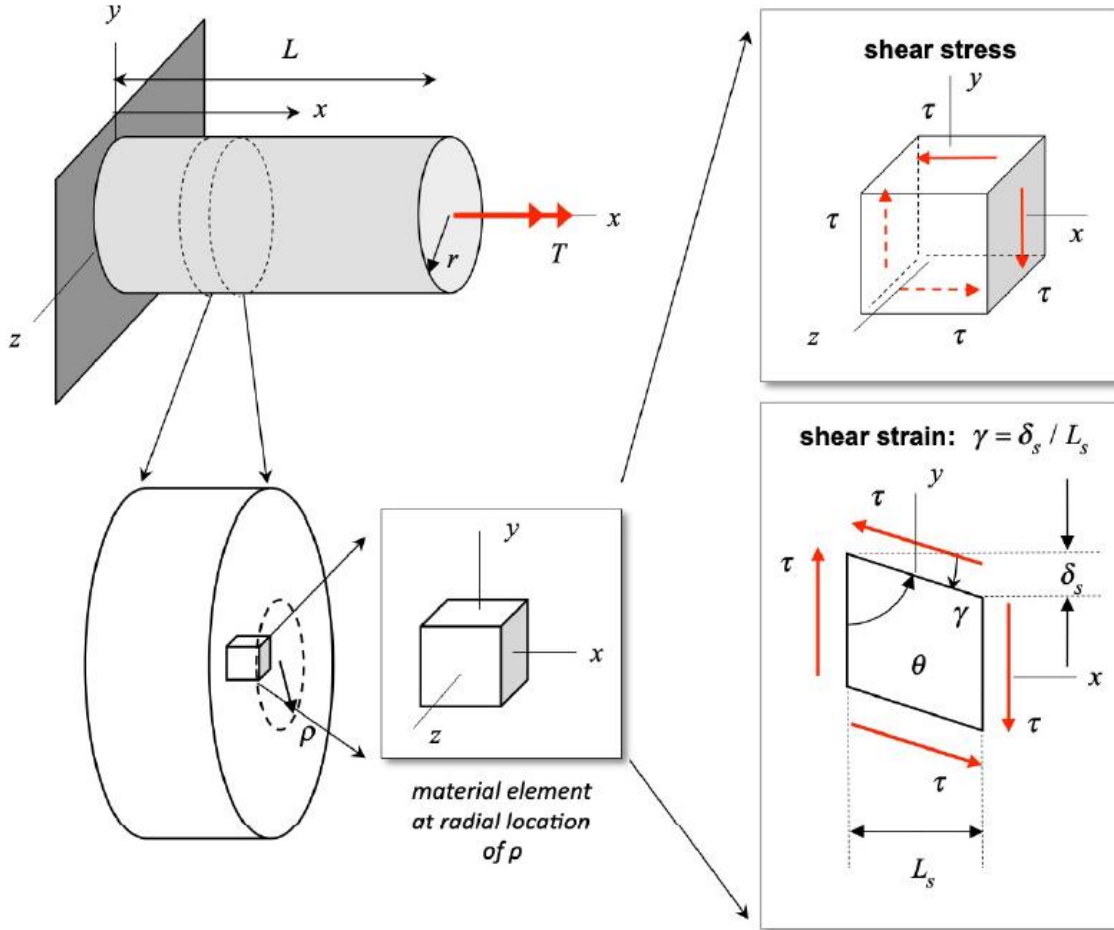
Shear strain and angle of twist; shear strain and shear stress

Lecture Book: Chapter 8, Page 4



Shear stress and torque

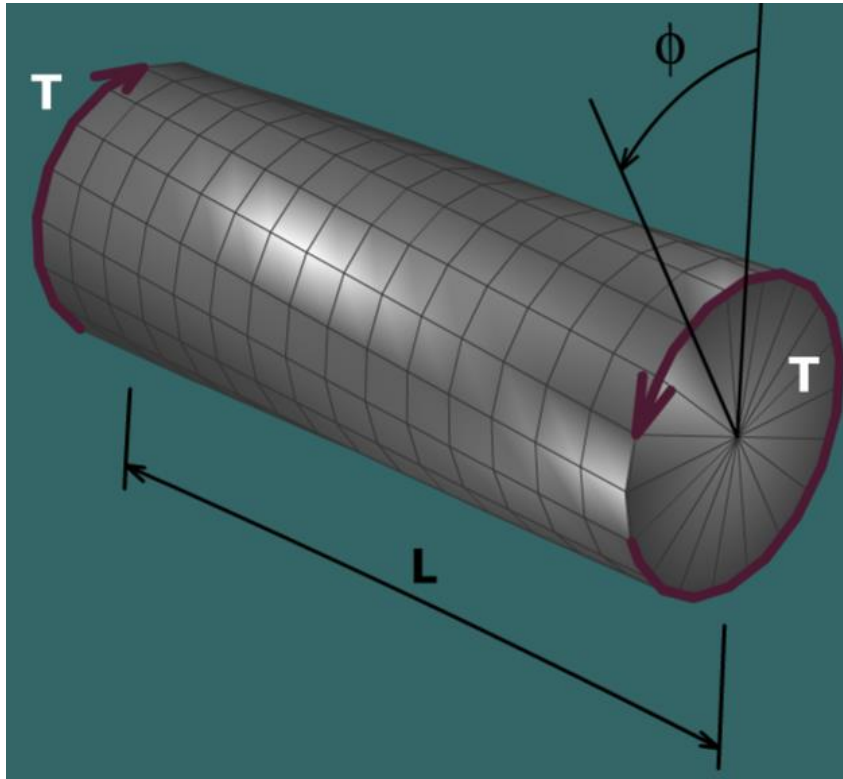
Lecture Book: Chapter 8, Pages 5-6



Torque-angle of twist relation

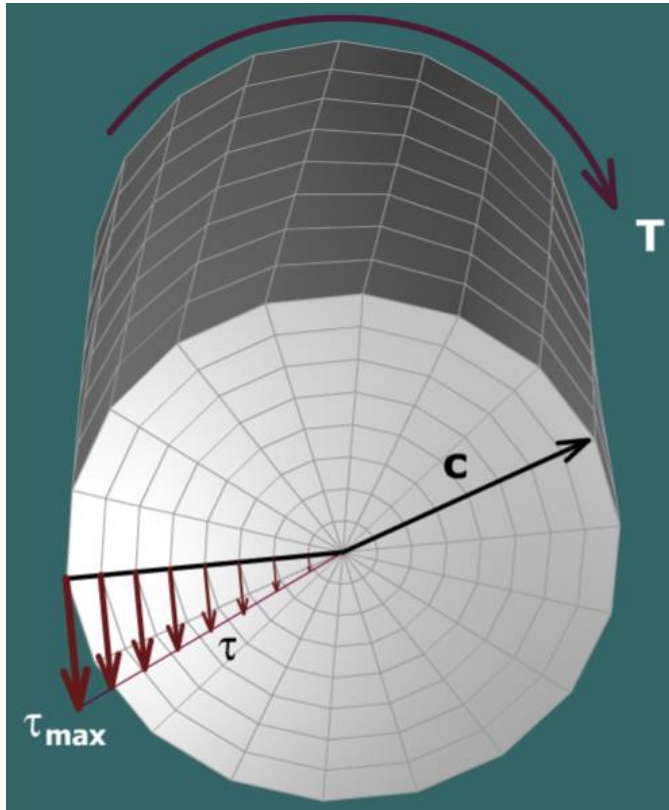
Using all of the previous equations, we can relate the torque T to the angle of twist ϕ

Lecture Book: Chapter 8, Page 6



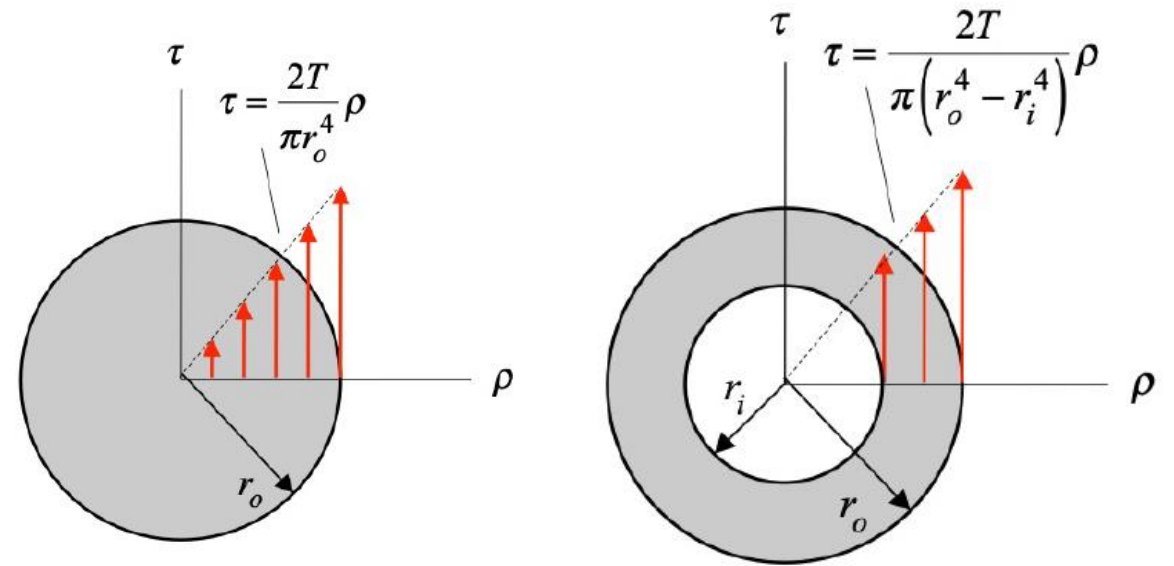
Notes on shear stress

For constant G throughout the cross section, the shear stress τ varies linearly with the radial distance ρ



Lecture Book: Chapter 8, Page 7

We will consider both solid and hollow (tubular) shafts

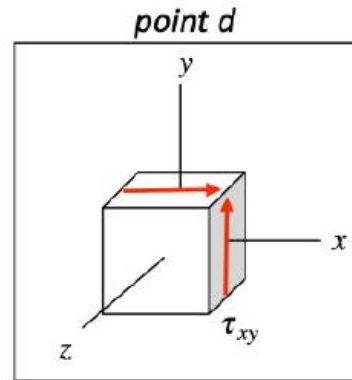
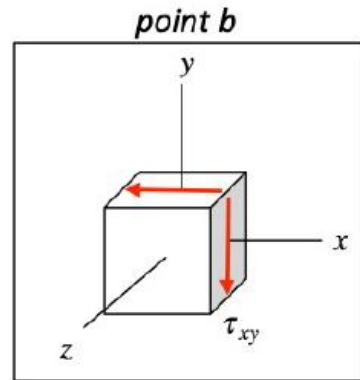
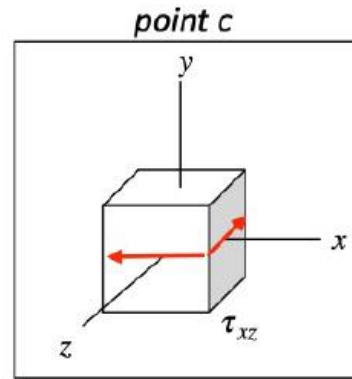
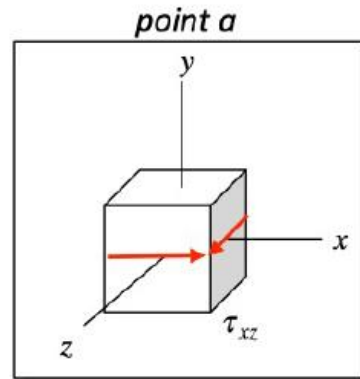
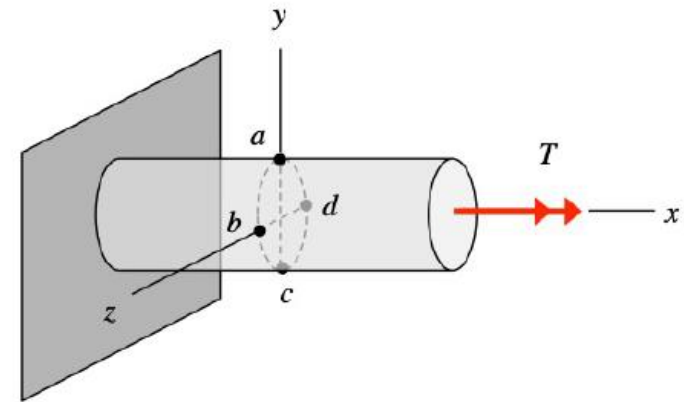


Notes on shear stress

States of stress at various points on a cross section—in all cases, what is the magnitude of τ ?

Lecture Book: Chapter 8, Page 8

state of stress at four locations on shaft face



Summary: Comparison with uniaxial deformation

Normal strain-displacement

$$\varepsilon = \frac{du}{dx}$$

Stress-strain (uniaxial stress)

$$\sigma = E\varepsilon$$

Axial force-normal stress

$$\sigma = \frac{F}{A}$$

Force-deformation (elongation)

$$e = u(L) - u(0) = \int_0^L \frac{F}{AE} dx$$

Constant F, A, E :
$$e = \frac{FL}{AE}$$

Shear strain-angle of twist

$$\gamma = \rho \frac{d\phi}{dx}$$

Stress-strain (shear)

$$\tau = G\gamma$$

Torque-shear stress

$$\tau = \frac{T\rho}{I_p} \quad \tau_{\max} = \frac{Tr}{I_p}$$

Torque-twist

$$\Delta\phi = \phi(L) - \phi(0) = \int_0^L \frac{T}{GI_p} dx$$

Constant T, I_p, G :
$$\Delta\phi = \frac{TL}{GI_p}$$

Summary: Comparison with uniaxial deformation

Uniaxial loading procedure:

1. FBDs and equilibrium
2. Force-deformation equations
3. Compatibility equation(s)
4. Solve for unknowns

Torsional loading procedure:

1. FBDs and equilibrium
2. Torque-twist equations
3. Compatibility equation(s)
4. Solve for unknowns