

**c) Generalized Hooke's law for normal stresses/strains**

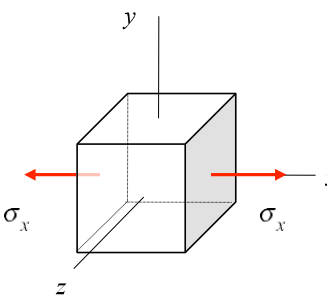
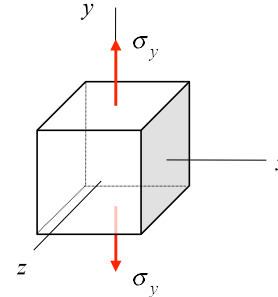
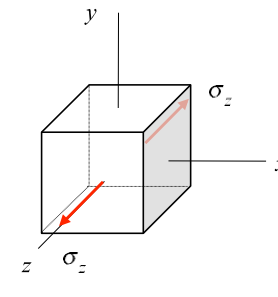
Recall that for uni-axial loading along the x-axis, the normal strains in the x, y and z directions in the body were found to be:

$$\epsilon_x = \sigma_x / E$$

$$\epsilon_y = \epsilon_z = -\nu\epsilon_x = -\nu\sigma_x / E$$

where  $E$  and  $\nu$  are the Young's modulus and Poisson's ratio for the material. For a 3-D loading of a body, we have three normal stress components  $\sigma_x$ ,  $\sigma_y$  and  $\sigma_z$  acting simultaneously. For this case, we will consider the strains due to each normal component of stress individually and add these together using linear superposition (along with the thermal strains) to determine the resulting three components of strain  $\epsilon_x$ ,  $\epsilon_y$  and  $\epsilon_z$ .

Consider the individual contributions of the three components of stress shown below:

Strains due to mechanical loading in the x-direction	Strains due to mechanical loading in the y-direction	Strains due to mechanical loading in the z-direction
		
$\epsilon_x = \sigma_x / E$	$\epsilon_x = -\nu\epsilon_y = -\nu\sigma_y / E$	$\epsilon_x = -\nu\epsilon_z = -\nu\sigma_z / E$
$\epsilon_y = -\nu\epsilon_x = -\nu\sigma_x / E$	$\epsilon_y = \sigma_y / E$	$\epsilon_y = -\nu\epsilon_z = -\nu\sigma_z / E$
$\epsilon_z = -\nu\epsilon_x = -\nu\sigma_x / E$	$\epsilon_z = -\nu\epsilon_y = -\nu\sigma_y / E$	$\epsilon_z = \sigma_z / E$

The total strain in each direction is found through superposition of the individual strains along with the thermal strains. Adding together these components (across each row of the above table) gives:

$$\epsilon_x = \frac{1}{E}\sigma_x - \frac{\nu}{E}\sigma_y - \frac{\nu}{E}\sigma_z + \alpha\Delta T = \frac{1}{E}\left[\sigma_x - \nu(\sigma_y + \sigma_z)\right] + \alpha\Delta T$$

$$\epsilon_y = -\frac{\nu}{E}\sigma_x + \frac{1}{E}\sigma_y - \frac{\nu}{E}\sigma_z + \alpha\Delta T = \frac{1}{E}\left[\sigma_y - \nu(\sigma_x + \sigma_z)\right] + \alpha\Delta T$$

$$\epsilon_z = -\frac{\nu}{E}\sigma_x - \frac{\nu}{E}\sigma_y + \frac{1}{E}\sigma_z + \alpha\Delta T = \frac{1}{E}\left[\sigma_z - \nu(\sigma_x + \sigma_y)\right] + \alpha\Delta T$$

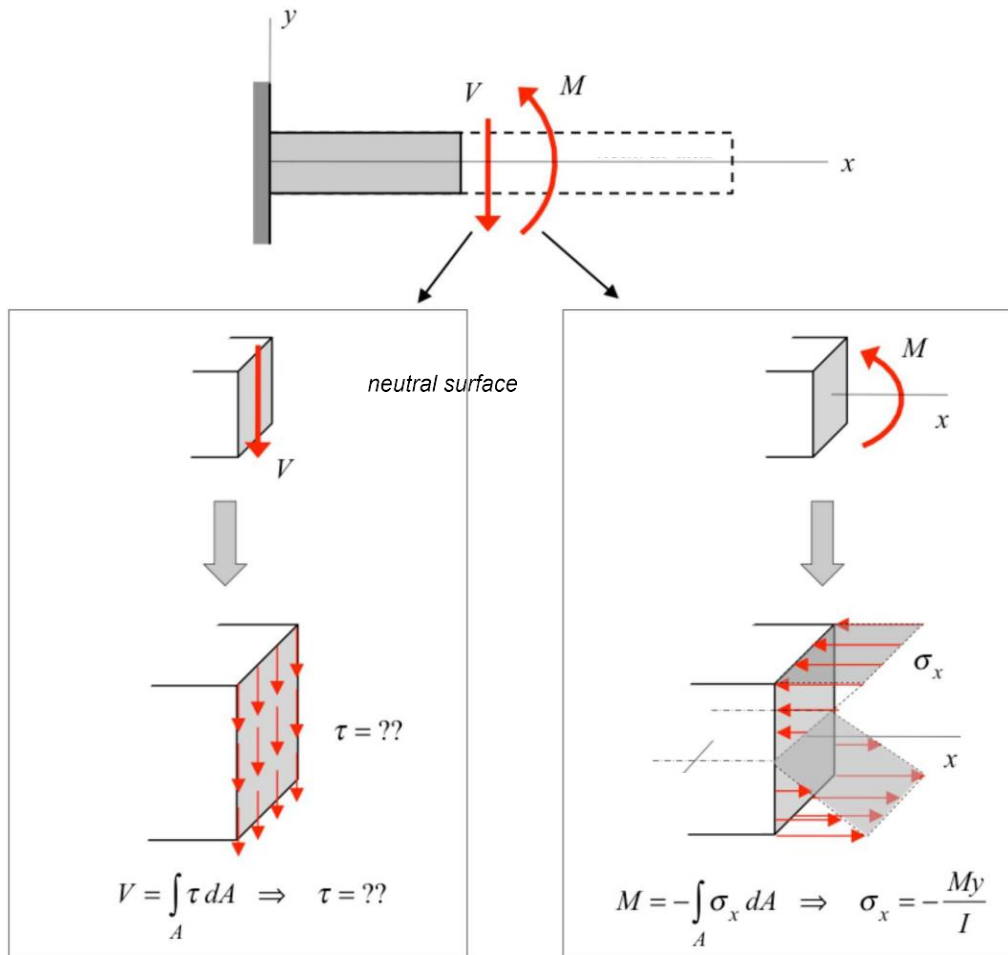
The above are known as the generalized Hooke's law equations for normal stresses/strains due to 3-D loadings on a body.

### **Summary: Indeterminate axial problems (the four-step method)**

1. *Draw FBD's and write down equilibrium equations.*
  - a. Break system into "elements". For problems in this course, an element:
    - i. has forces acting only at its ends,
    - ii. has a constant cross section, and
    - iii. has constant material properties.
  - b. It is recommended that you always draw end forces on element corresponding to "tension" (if the force actually corresponds to compression, you will get a negative value for the force in the end...trust the math...it works!).
  - c. Be sure to abide by Newton's 3rd Law (reactions appear in equal and opposite pairs) when drawing your FBD's
  
2. *Write down the elemental force-deformation equations.*
  - a. For the  $j$ th element:
$$e_j = \frac{P_j L_j}{E_j A_j}$$
  - b. If you draw all element forces as in tension (as recommended in 1b) above), then  $P_j$  has a positive sign in the equation above. If you choose to draw the elemental force as in compression, then  $P_j$  has a negative sign.
  
3. *Write down appropriate compatibility equations*
  - a. In this step you will write down the constraint equations that exists among the element elongations.
  - b. This step is *problem-dependent* (and requires the most thought):
    - i. For an axial system constrained between rigid supports, the compatibility equation needed is that the sum of all the elemental elongations is zero.
    - ii. For elements attached to a rigid member, the motion of the rigid member dictates the relationship that exists among the elemental elongations.
    - iii. For truss elements, a trigometric relationship must be used to relate the elemental elongations.
  
4. *Solve equations derived in Steps 1-3 for the elemental forces  $P_j$ .* Count your number of equations and number of unknowns. If you have sufficient equations to solve, then solve. If not, review the first three steps to see if you are missing needed equilibrium, force/deformation or compatibility equations. From these forces determine at this step, the elemental stresses, strains and elongations can be computed as needed.

c) Stresses due general transverse force and bending-couple loading of beams

Earlier in the chapter, we considered the normal stress distribution within the cross section of a beam experiencing pure bending (i.e., in the absence of a shear force resultant on the cross-sectional cut). Here we will now consider the more general case of having both shear force and bending moment couples on the cross-sectional cut, as demonstrated by the figure below.



**PROBLEM #3 (25 points)**

A torque  $T$  is applied to a gear-shaft system and is transmitted through rigid gears B and C to a fixed end E as shown in Fig. 3(a). The shafts (1) and (2) are tightly fit to each other. Frictionless bearings are used to support the shafts. The geometry and material property of the shafts and gears are listed in the following table.

	Size	Length	Shear modulus
Shaft (1)	Outer diameter = $2d$ Inner diameter = $d$	$L$	$2G$
Shaft (2)	Diameter = $d$	$L$	$G$
Shaft (3)	Diameter = $d$	$L/2$	$G$
Shaft (4)	Diameter = $d$	$L$	$G$
Gear B	Diameter = $1.5d$	Negligible	Rigid
Gear C	Diameter = $3d$	Negligible	Rigid

- (a) Determine the torque carried by each shaft.
- (b) Determine the angle of twist at the free ends A and D.
- (c) Consider the cross section  $aa'$  for the shafts (1) and (2), show the magnitude of the shear stress as a function of the distance from the center on Fig. 3(b). Mark the critical values in the diagram.
- (d) Consider the points M and N on the cross section  $aa'$ , shown in Fig. 3(c). Sketch the stress states at M and N on the stress elements on Fig. 3(d).

Express all your answers in terms of  $d, L, G, T, \pi$ .

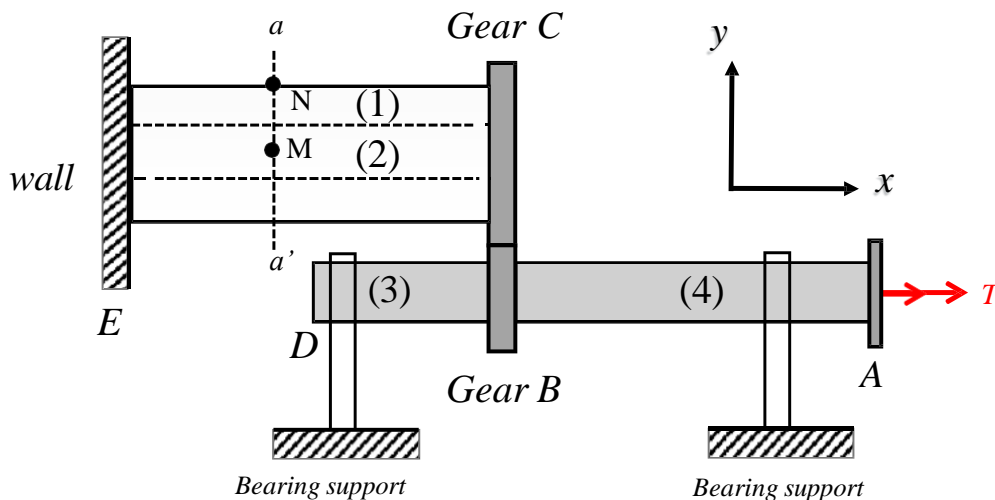


Fig. 3(a)

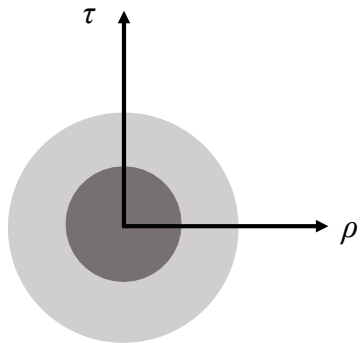


Fig. 3(b)

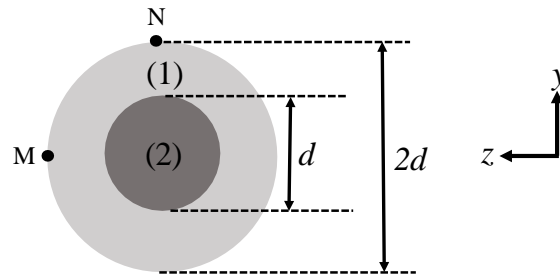


Fig. 3(c)

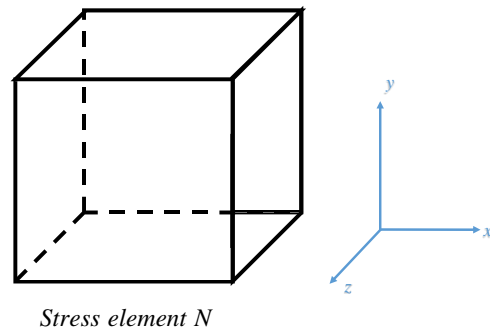
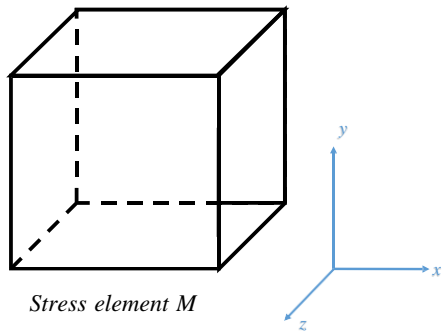


Fig. 3(d)

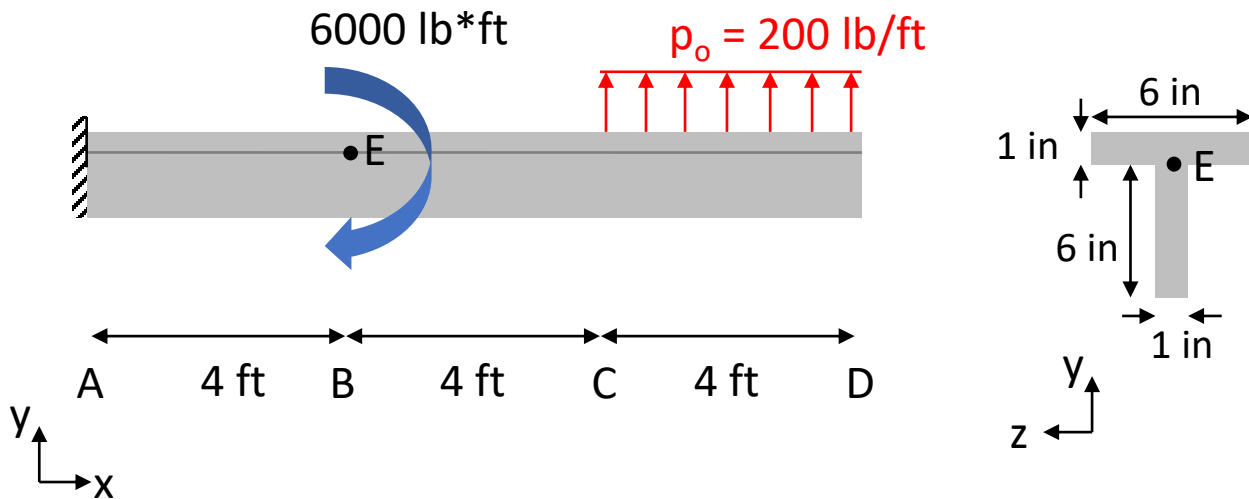
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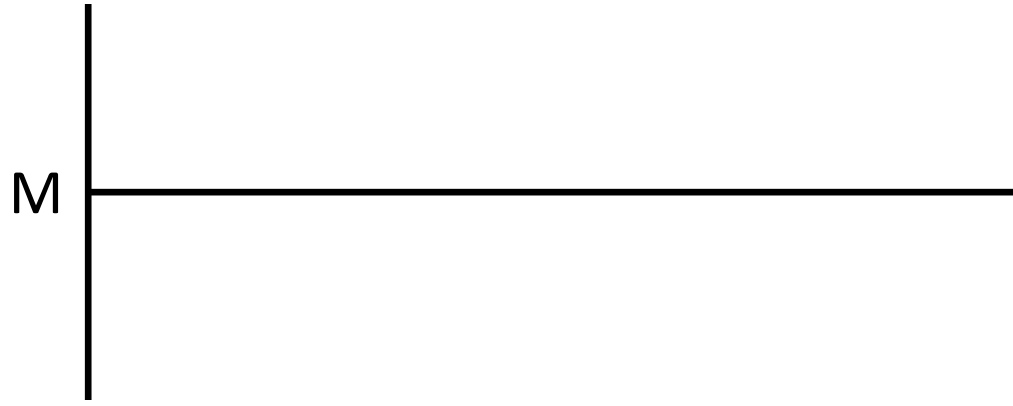
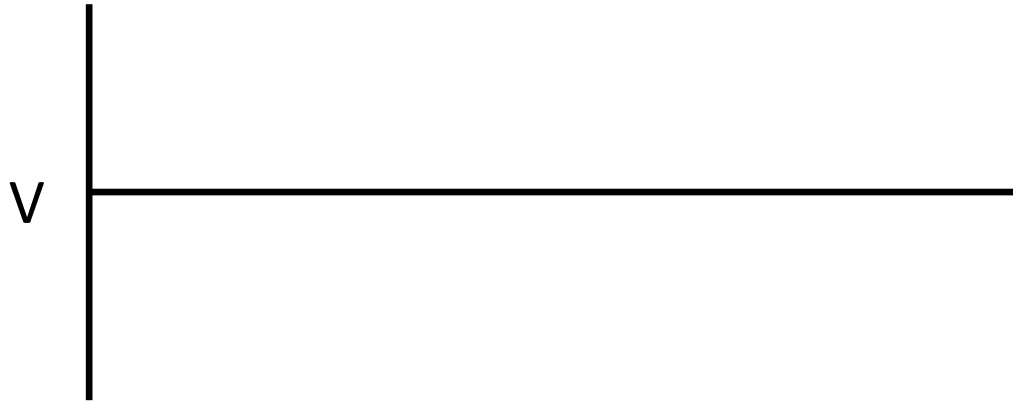
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# Exam 1 Example Question



- Draw the shear force and bending moment diagrams.
- For the cross-section with the largest normal stress, draw the profile of the normal stress on the cross-section. Include the numerical values of the maximum and minimum normal stresses.
- What is the shear stress at point E<sup>-</sup>? What is the shear stress at point E<sup>+</sup>?
- Draw the profile of the shear stress on the cross-section. Include numerical values for the key points.







**PROBLEM #1 (25 points)**

Rigid bracket CAD is supported by elastic rods (1) and (2). A force  $P$  is applied downward at point B. Simultaneously, the temperature of rod (2) is changed by an amount  $\Delta T$ . The temperature of rod (1) is kept constant. The geometry and material properties of rods (1) and (2) are listed in the following table:

	Cross-sectional area	Length	Young's modulus	Coefficient of thermal expansion
Rod (1)	$A$	$L$	$2E$	$\alpha$
Rod (2)	$A$	$1.5L$	$E$	$\alpha$

- (a) If the displacement of point C is known to be  $10^{-4}$  in. downward, determine the axial force  $F_1$  carried by rod (1), the axial force  $F_2$  carried by rod (2), and the applied force  $P$ .
- (b) Using the forces determined in part (a), determine the factor of safety guarding against yielding of pin A if the pin has a cross-sectional area of  $0.1 \text{ in}^2$ , shear yield strength  $\tau_Y = 20 \text{ ksi}$ , and is connected to the ground by a double-sided connection as shown in Figure 1.b.

Use the following numerical values:  $E = 30 \times 10^3 \text{ ksi}$ ,  $\alpha = 10^{-6} / ^\circ\text{F}$ ,  $L = 10 \text{ in}$ ,  $A = 2 \text{ in}^2$ ,  $\Delta T = -20 \text{ }^\circ\text{F}$  (i.e. the temperature of rod (2) is decreased)

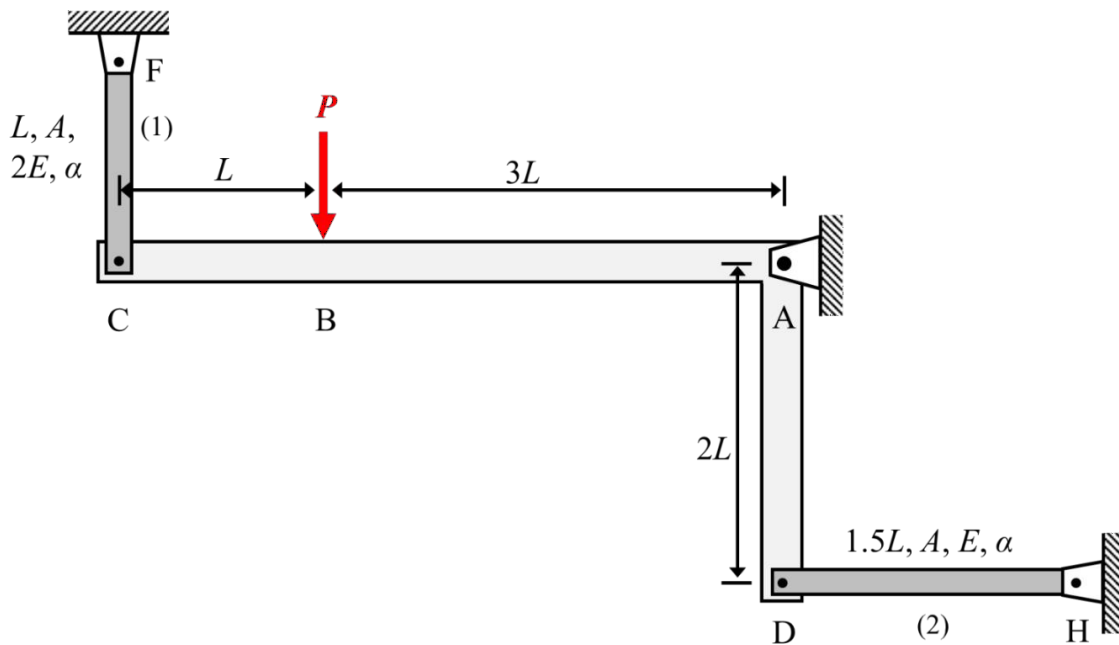


Figure 1.a.

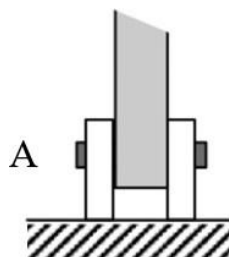


Figure 1.b: Edge view of the pinned connection at A.

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# Fall 2019 Exam 1

ME 323 Examination # 1

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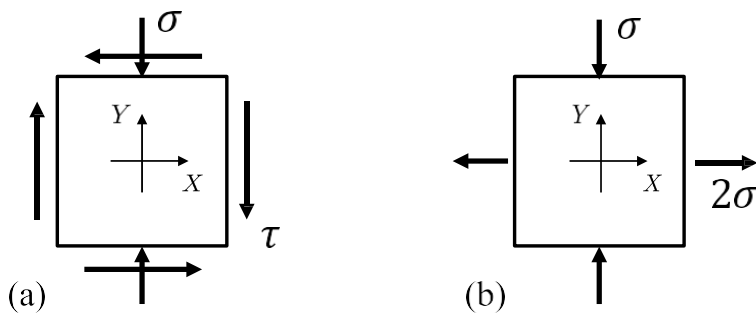
**PROBLEM #4 (25 Points):**

**PART A – 6 points**

For each state of plane stress shown below, i.e., for configurations (a) and (b), indicate whether each component of the state of strain is:

- ❖ = 0 (equal to zero)
- ❖ > 0 (greater than zero)
- ❖ < 0 (less than zero)

The material is linear elastic with Poisson’s ratio  $\nu$  ( $0 < \nu < 0.5$ ), and the deformations are small.



	(a)	(b)
$\epsilon_x$		
$\epsilon_y$		
$\epsilon_z$		
$\gamma_{xy}$		
$\gamma_{xz}$		
$\gamma_{yz}$		

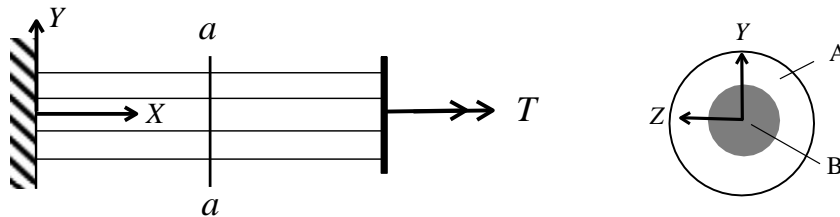
Fill in with ‘= 0’, ‘> 0’, or ‘< 0’.

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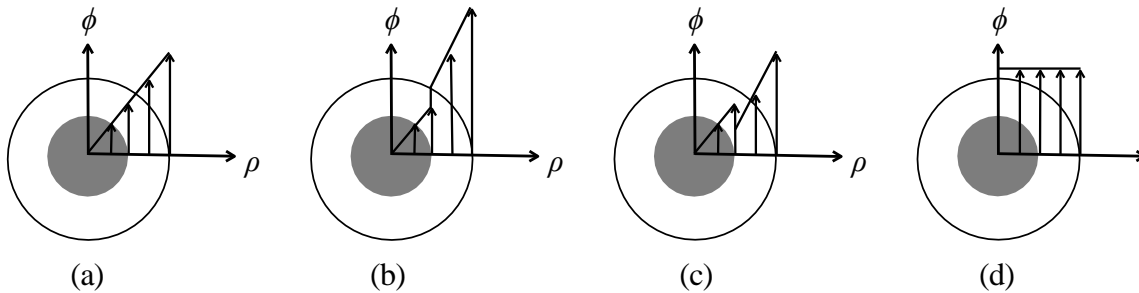
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**4.2. (6 Points)**

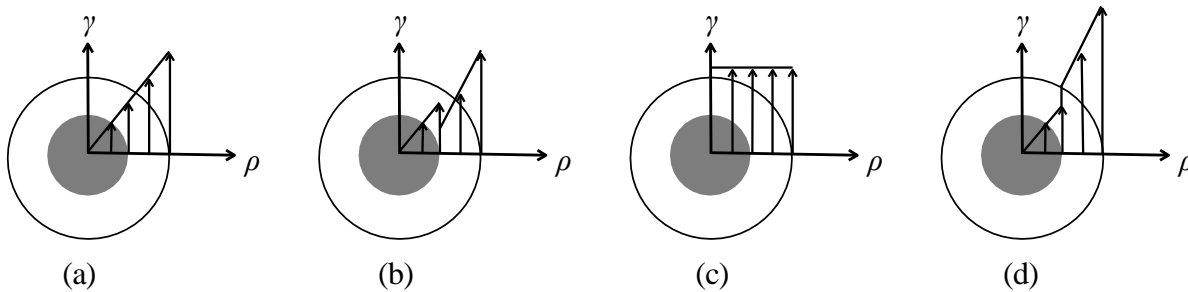
A bimetallic bar with circular cross section consists of a shell A and a core B. The bimetallic bar is subjected to a torque  $T$ . The shear moduli of the core and shell are known to be  $G_A = 2G_B$ , and polar moment of inertia  $I_{PA} = 0.5 I_{PB}$ .



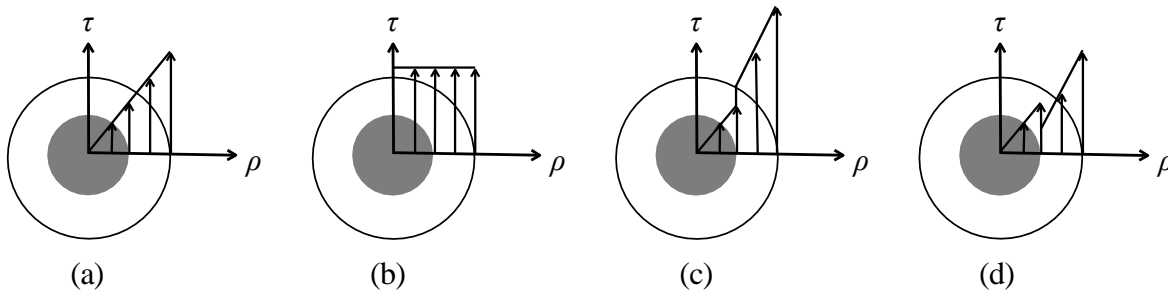
Which figure shows the correct distribution of the twist angle in the cross section  $aa$ ?



Which figure shows the correct distribution of the shear strain in the cross section  $aa$ ?

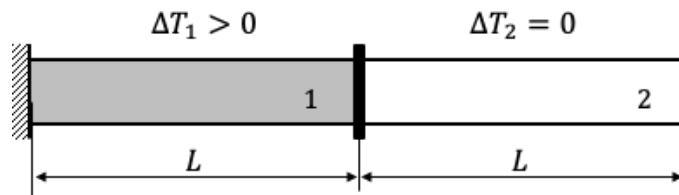


Which figure shows the correct distribution of the shear stress in the cross section  $aa$ ?



**PROBLEM #4 (cont.):****PART B – 4 points**

A rod is made up of elastic elements 1 and 2, each having a length  $L$  and cross-sectional area  $A$ . Elements 1 and 2 have the same Young's modulus  $E_1 = E_2$  and coefficient of thermal expansion  $\alpha_1 = \alpha_2$ . Let  $F_1$  and  $F_2$  represent the axial load carried by elements 1 and 2, respectively, when the temperature of element 1 is increased by  $\Delta T_1 > 0$  — while the temperature of element 2 is kept constant  $\Delta T_2 = 0$ .



Circle the correct answer:

- a)  $|e_1| > |e_2|$
- b)  $|e_1| = |e_2|$
- c)  $|e_1| < |e_2|$

Circle the correct answer:

- a)  $|F_1| > |F_2|$
- b)  $|F_1| = |F_2|$
- c)  $|F_1| < |F_2|$

Circle the correct answer:

- a) Elastic element 1 is under tension
- b) Elastic element 1 is under compression
- c) Elastic element 1 is stress-free

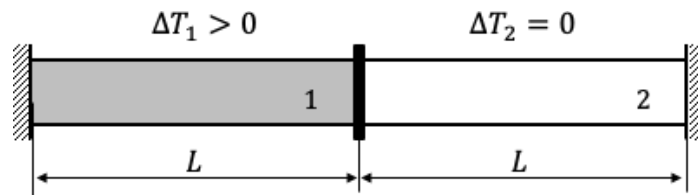
Circle the correct answer:

- a) Elastic element 2 is under tension
- b) Elastic element 2 is under compression
- c) Elastic element 2 is stress-free



**PROBLEM #4 (cont.):****PART C – 4 points**

A rod is made up of elastic elements 1 and 2, each having a length  $L$  and cross-sectional area  $A$ . Elements 1 and 2 have the same Young's modulus  $E_1=E_2$  and coefficient of thermal expansion  $\alpha_1=\alpha_2$ . Let  $F_1$  and  $F_2$  represent the axial load carried by elements 1 and 2, respectively, when the temperature of element 1 is increased by  $\Delta T_1 > 0$  — while the temperature of element 2 is kept constant  $\Delta T_2 = 0$ .



Circle the correct answer:

- a)  $|e_1| > |e_2|$
- b)  $|e_1| = |e_2|$
- c)  $|e_1| < |e_2|$

Circle the correct answer:

- a)  $|F_1| > |F_2|$
- b)  $|F_1| = |F_2|$
- c)  $|F_1| < |F_2|$

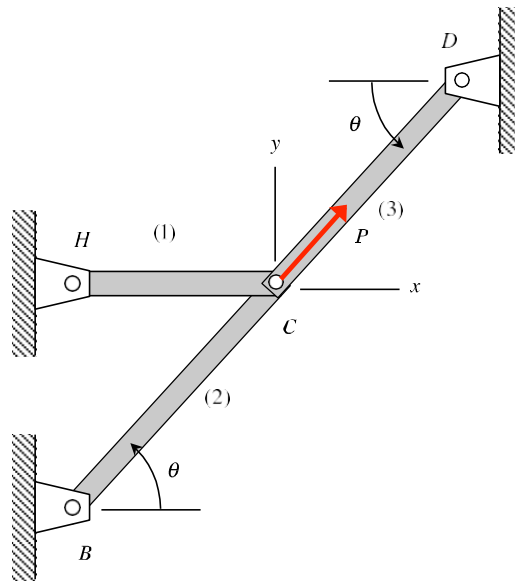
Circle the correct answer:

- a) Elastic element 1 is under tension
- b) Elastic element 1 is under compression
- c) Elastic element 1 is stress-free

Circle the correct answer:

- a) Elastic element 2 is under tension
- b) Elastic element 2 is under compression
- c) Elastic element 2 is stress-free

**PROBLEM NO. 4 - PART C – 6 points max.**



In the truss shown above, member (1) is horizontal, with members (2) and (3) aligned and at an angle of  $\theta$  with respect to the horizontal. A load  $P$  is applied to joint  $C$  in a direction that is aligned with members (2) and (3). Simultaneously, the temperature of member (2) is *increased*, with the temperatures of the remaining members being held constant. Let  $e_1$  be the elongation of member (1), and  $(u_C, v_C)$  being the  $x$ - and  $y$ -components of displacement of joint  $C$  due to the load  $P$ .

For this loading on the truss, the axial stress in member (1) is (circle the correct response):

- a) compressive.
- b) tensile.
- c) zero.

*HINT:* consider an FBD of joint  $C$ .

Also, for this loading the *displacement* of joint  $C$  is (circle the correct response):

- a) up and to the right ( $u_C > 0$  and  $v_C > 0$ )
- b) directly to the right ( $u_C > 0$  and  $v_C = 0$ )
- c) directly up ( $u_C = 0$  and  $v_C > 0$ )
- d) zero ( $u_C = 0$  and  $v_C = 0$ )

You are NOT asked to provide explanations for your answers.

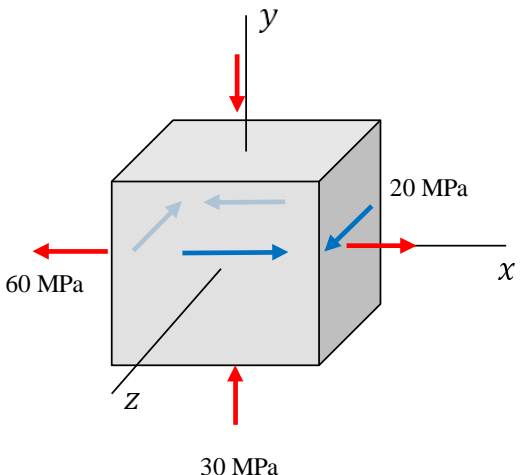
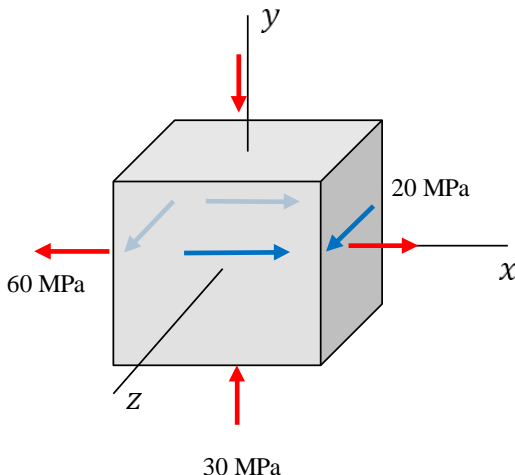
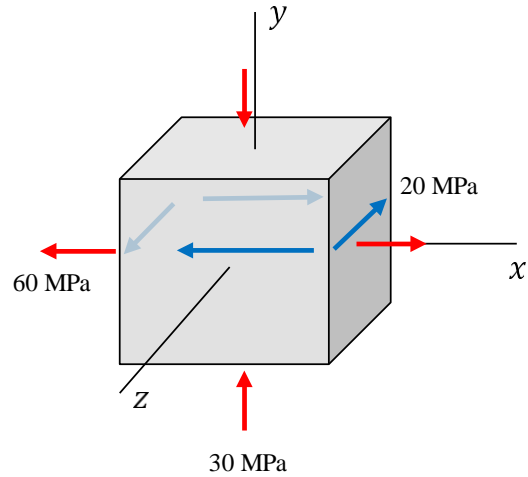
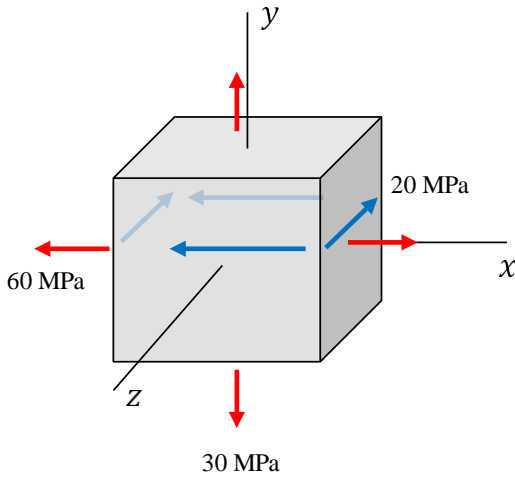
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**PROBLEM #4 (24 Points):**

**4.1. (8 Points)**

A material point in a steel machine is subjected to the following stress state:  $\sigma_x = 60 \text{ MPa}$ ,  $\sigma_y = -30 \text{ MPa}$ , and  $\tau_{xz} = -20 \text{ MPa}$ . Which of the following stress elements represents the correct stress state of the material point?



Using  $E=210 \text{ GPa}$  and  $\nu=0.3$  for steel,  $G = \frac{E}{2(1+\nu)}$ , determine the strain components:

$\epsilon_x =$  \_\_\_\_\_

$\epsilon_y =$  \_\_\_\_\_

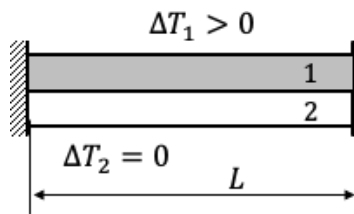
$\epsilon_z =$  \_\_\_\_\_

$\gamma_{xy} =$  \_\_\_\_\_

$\gamma_{yz} =$  \_\_\_\_\_

**PROBLEM #4 (cont.):****PART D – 4 points**

A rod is made up of elastic elements 1 and 2, each having a length  $L$  and cross-sectional area  $A$ . Elements 1 and 2 have the same Young's modulus  $E_1=E_2$  and coefficient of thermal expansion  $\alpha_1=\alpha_2$ . Let  $F_1$  and  $F_2$  represent the axial load carried by elements 1 and 2, respectively, when the temperature of element 1 is increased by  $\Delta T_1 > 0$  — while the temperature of element 2 is kept constant  $\Delta T_2 = 0$ .



Circle the correct answer:

- a)  $|e_1| > |e_2|$
- b)  $|e_1| = |e_2|$
- c)  $|e_1| < |e_2|$

Circle the correct answer:

- a)  $|F_1| > |F_2|$
- b)  $|F_1| = |F_2|$
- c)  $|F_1| < |F_2|$

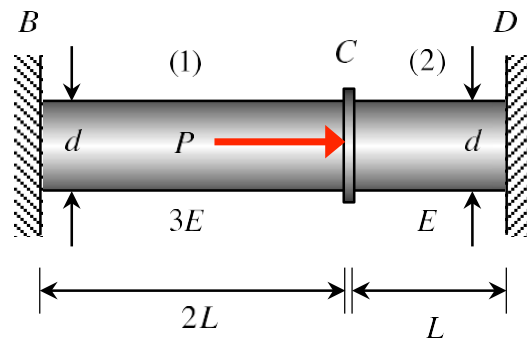
Circle the correct answer:

- a) Elastic element 1 is under tension
- b) Elastic element 1 is under compression
- c) Elastic element 1 is stress-free

Circle the correct answer:

- a) Elastic element 2 is under tension
- b) Elastic element 2 is under compression
- c) Elastic element 2 is stress-free

**PROBLEM NO. 4 - PART B – 3 points max.**

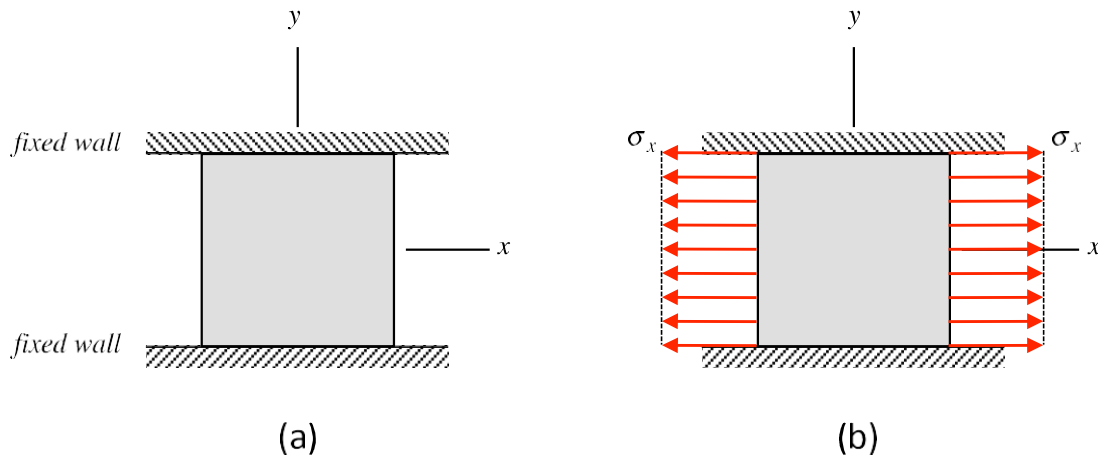


A rod is made up of elements (1) and (2) joined together by the rigid connector at D, with the elements having solid circular cross sections. The materials of elements (1) and (2) have Young's moduli of  $E_1 = 3E$  and  $E_2 = E$ , respectively. As a result of the axial load  $P$  applied at connector C, members (1) and (2) carry loads of  $F_1$  and  $F_2$ , respectively. Circle the item below that most accurately describes the relative sizes of the load magnitudes in the two elements:

- a)  $|F_1| > |F_2|$
- b)  $|F_1| = |F_2|$
- c)  $|F_1| < |F_2|$
- d) More information is needed in order to answer this question.

You are NOT asked to provide an explanation for your answer.

**PROBLEM NO. 4 - PART D** – 6 points max.



A square homogeneous block made up of a material with a Poisson's ratio of  $\nu = 0.3$  is placed between two smooth, rigid walls. Initially, the temperature of the block in Figure (a) above is increased by an amount that produces a compressive normal stress of  $\sigma_y = -20 \text{ ksi}$ . After that, the block is given an additional tensile stress component  $\sigma_x$ , as shown in Figure (b) above, with this stress, in turn, reducing the y-component of stress to  $\sigma_y = -5 \text{ ksi}$ . Determine the value of  $\sigma_x$ .