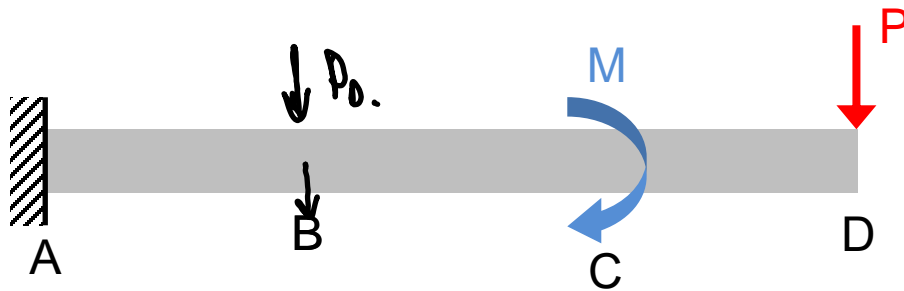


## Castigliano's Review

Question 1:

In the beam, the displacement at B is found by:

- The partial derivative of U with respect to P.
- The partial derivative of U with respect to M.
- ~~The partial derivative of U with respect to a dummy load.~~
- The partial derivative of U with respect to a dummy moment.



$$\frac{\partial U}{\partial P} = \Delta$$

**Example 16.8**

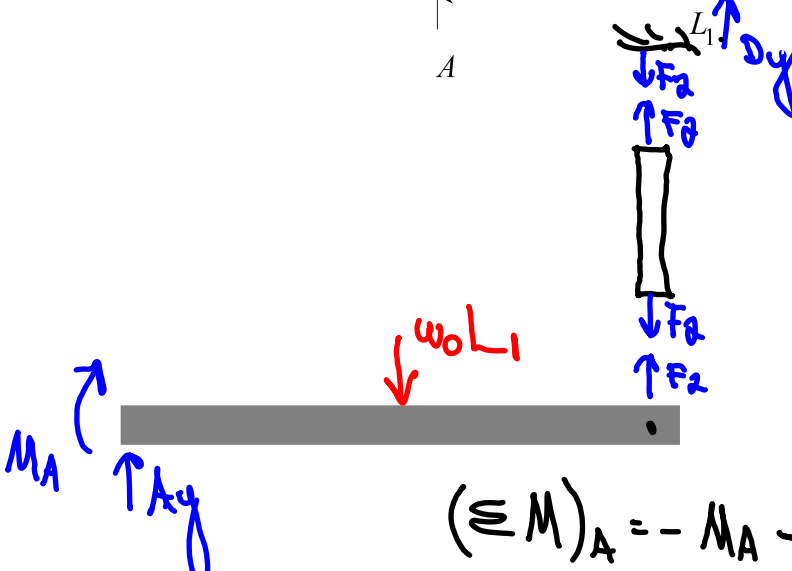
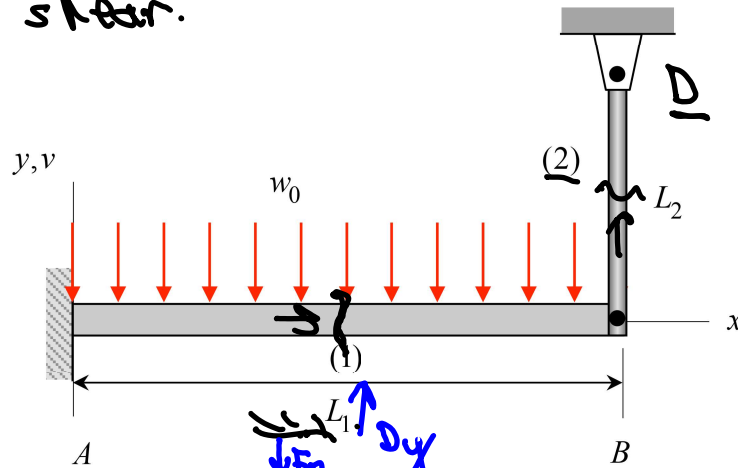
For the following examples, set up the problem for determining the requested deflections using Castigliano's method:

- draw appropriate FBDs;
- determine internal results for each section;
- set up the integrals for calculating the required deflections;
- explain how Castigliano's method is used to solve. Discuss the application of dummy forces (when needed) and how to handle redundant forces for indeterminate structures.

**Problem A**

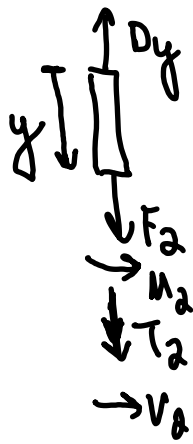
Find the load carried by member (2) of the structure below. Let  $E$  and  $A$  be the Young's modulus and cross-sectional area, respectively, of member (2), whereas  $E$  and  $I$  are the Young's modulus and second area moment of the cross section of (1), respectively.

Don't neglect shear.



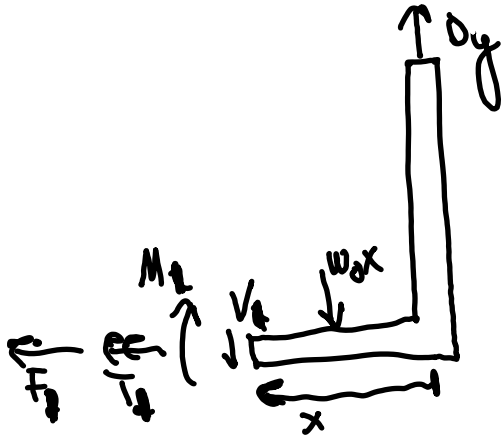
$$\left. \begin{aligned} (\sum M)_A &= -M_A - w_0 L \left(\frac{L}{2}\right) + F_2 L = 0 \\ (\sum F_y) &= A_y - w_0 L + F_2 = 0 \end{aligned} \right\} \begin{array}{l} 3 \text{ unknowns} \\ 2 \text{ eqns} \\ \hline 1 \end{array}$$

$$(\sum F_y)_0 = F_2 = 0y$$



$$\sum F_y = D_y - F_2 = 0$$

$$F_2(y) = D_y$$



$$\sum F_y = D_y - V_2 - w_0x = 0$$

$$V_2(x) = D_y - w_0x$$

$$(\sum M)_2 = D_yx - w_0x\left(\frac{x}{2}\right) - M_1 = 0$$

$$M_1(x) = D_yx - w_0\frac{x^2}{2}$$

$$U = U_{M_1} + U_{V_1} + U_{F_2}$$

$$U = \frac{1}{2EI} \int_0^{L_1} M_1^2 dx + \frac{f_s}{2GA} \int_0^{L_1} V_1^2 dx + \frac{1}{2EA} \int_0^{L_2} F_1^2 dy$$

$$\frac{\partial U}{\partial D_y} = 0$$

$$\frac{\partial F_2}{\partial D_y} = 1$$

$$\frac{\partial M_2}{\partial D_y} = x$$

$$\frac{\partial V_1}{\partial D_y} = 1$$

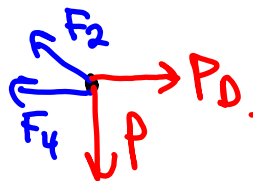
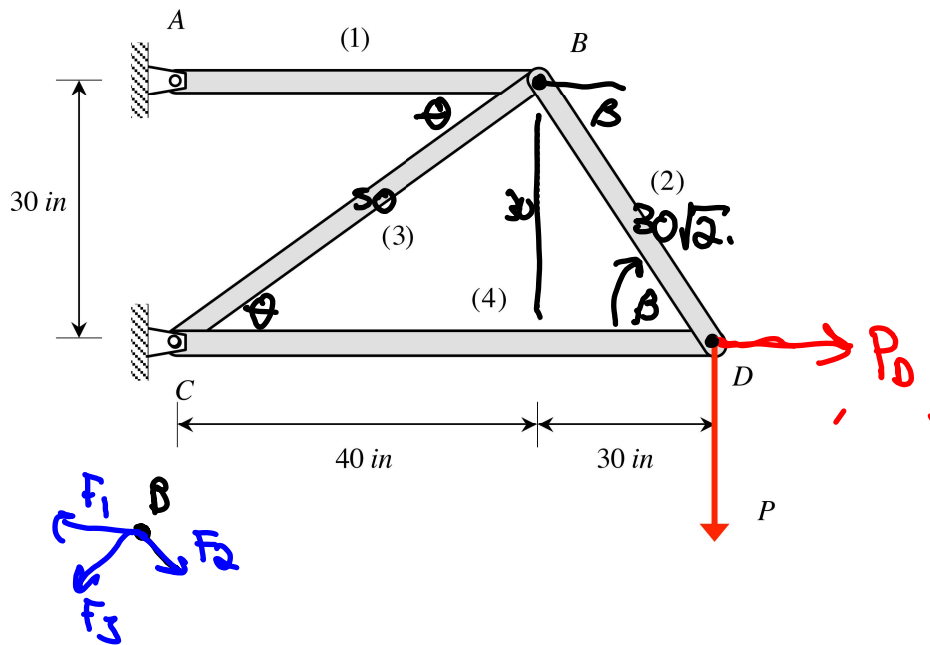
$$\frac{\partial U}{\partial D_y} = 0 = \frac{1}{EI} \int_0^{L_1} (D_yx - w_0\frac{x^2}{2})x dx + \frac{f_s}{GA} \int_0^{L_1} (D_y - w_0x) dx + \frac{1}{EA} \int_0^{L_2} D_y dy$$

$$\left[ \frac{D_y L_2}{EA} \right] = e_2$$

$$-e = \frac{1}{EI} \int_0^L (Dy - w_0 \frac{x^2}{2}) x dx + \frac{F_3}{GA} \int_0^{L_1} (Dy - w_0 x) dx.$$

**Problem D**

Determine the vertical and horizontal deflection of the truss at joint D. All members of the truss have a cross-sectional area of  $A$  and are made of a material with a Young's modulus of  $E$ .



$$\sum F_x)_B = -F_1 - F_3\left(\frac{4}{5}\right) + F_2\left(\frac{1}{\sqrt{2}}\right) = 0$$

$$\sum F_y)_B = -F_3\left(\frac{3}{5}\right) - F_2\left(\frac{1}{\sqrt{2}}\right) = 0$$

$$\sum F_x)_D = P_D - F_4 - F_2\left(\frac{1}{\sqrt{2}}\right) = 0$$

$$\sum F_y)_D = -P + F_2\left(\frac{1}{\sqrt{2}}\right) = 0$$

4 unknowns  
4 eqns.

$$F_2 = \sqrt{2} P$$

$$F_4 = P_0 - P \leftarrow$$

$$F_3 = -\left(\frac{5}{3}\right)P$$

$$F_1 = \frac{4}{3}P$$

$$U = U_{F_1} + U_{F_2} + U_{F_3} + U_{F_4}$$

$$u = \left[ \frac{\partial U}{\partial P_0} \right]_{P_0=0}$$

$$v = - \left[ \frac{\partial U}{\partial P} \right]_{P_0=0}$$

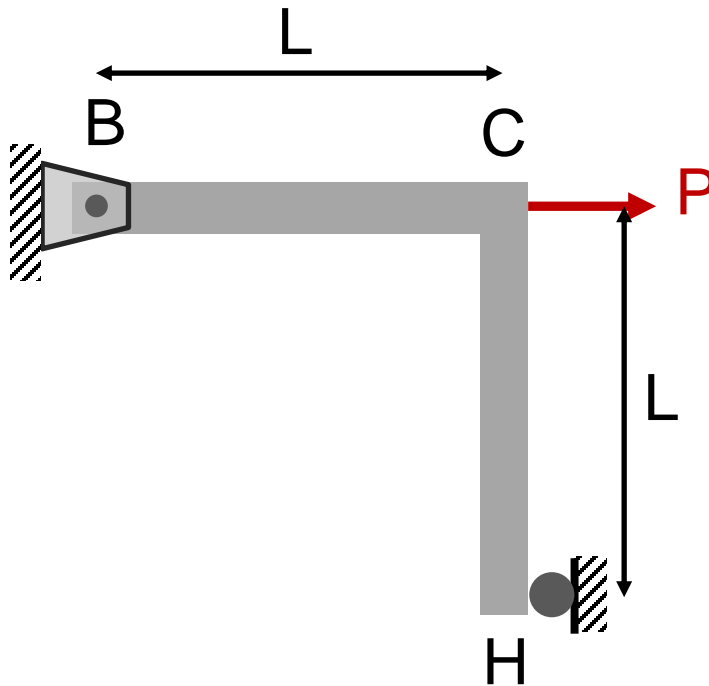
$$\frac{\partial F_1}{\partial P_0} = \frac{\partial F_2}{\partial P_0} = \frac{\partial F_3}{\partial P_0} = 0 \quad \frac{\partial F_4}{\partial P_0} = 1.$$

$$u = \frac{4}{EA} (-P)(1)$$



## Lecture 24 Quiz

An L-shaped beam is subject to an applied force as shown in the diagram below. You are asked to use Castigliano's to solve for the angular change at H ( $\theta_H$ ). Structure BCH has a modulus of E and a second area moment of I.



- Draw the free body diagram.
- Solve for the angular change at H.

Ch 16: 29.