7-4  $F \cos 20^{\circ}(d/2) = T_A$ ,  $F = 2 T_A/(d \cos 20^{\circ}) = 2(340)/(0.150 \cos 20^{\circ}) = 4824 \text{ N}$ . The maximum bending moment will be at point C, with  $M_C = 4824(0.100) = 482.4 \text{ N} \cdot \text{m}$ . Due to the rotation, the bending is completely reversed, while the torsion is constant. Thus,  $M_a = 482.4 \text{ N} \cdot \text{m}$ ,  $T_m = 340 \text{ N} \cdot \text{m}$ ,  $T_m = 340 \text{ N} \cdot \text{m}$ ,  $T_m = 340 \text{ N} \cdot \text{m}$ .

For sharp fillet radii at the shoulders, from Table 7-1,  $K_t = 2.7$ , and  $K_{ts} = 2.2$ . Examining Figs. 6-26 and 6-27 with  $S_{ut} = 560$  MPa, conservatively estimate q = 0.8 and  $q_s = 0.9$ . These estimates can be checked once a specific fillet radius is determined.

Eq. (6-32): 
$$K_f = 1 + 0.8(2.7 - 1) = 2.4$$
  
 $K_{fs} = 1 + 0.9(2.2 - 1) = 2.1$ 

(a) We will choose to include fatigue stress concentration factors even for the static analysis to avoid localized yielding.

Eq. (7-15): 
$$\sigma'_{\text{max}} = \left[ \left( \frac{32K_f M_a}{\pi d^3} \right)^2 + 3 \left( \frac{16K_{fs} T_m}{\pi d^3} \right)^2 \right]^{1/2}$$

Eq. (7-16): 
$$n = \frac{S_y}{\sigma'_{\text{max}}} = \frac{\pi d^3 S_y}{16} \left[ 4 \left( K_f M_a \right)^2 + 3 \left( K_{fs} T_m \right)^2 \right]^{-1/2}$$

Solving for *d*,

$$d = \left\{ \frac{16n}{\pi S_y} \left[ 4(K_f M_a)^2 + 3(K_{fs} T_a)^2 \right]^{1/2} \right\}^{1/3}$$
$$= \left( \frac{16(2.5)}{\pi (420)(10^6)} \left\{ 4\left[ (2.4)(482.4) \right]^2 + 3\left[ (2.1)(340) \right]^2 \right\}^{1/2} \right)^{1/3}$$

$$d = 0.0430 \text{ m} = 43.0 \text{ mm}$$
 Ans.

**(b)** Eq. (6-18): 
$$k_a = 3.04(560)^{-0.217} = 0.77$$

Assume  $k_b = 0.85$  for now. Check later once a diameter is known.  $S_e = 0.77(0.85)(0.5)(560) = 183$  MPa

Selecting the DE-Goodman criteria, use Eqs. (7-6) and (7-8) with  $M_m = T_a = 0$ .

$$d = \left(\frac{16n}{\pi} \left\{ \frac{1}{S_e} \left[ 4 \left( K_f M_a \right)^2 \right]^{1/2} + \frac{1}{S_{ut}} \left[ 3 \left( K_{fs} T_m \right)^2 \right]^{1/2} \right\} \right)^{1/3}$$

$$= \left( \frac{16(2.5)}{\pi} \left\{ \frac{1}{183(10^6)} \left[ 4 \left( 2.4 \times 482.4 \right)^2 \right]^{1/2} + \frac{1}{560(10^6)} \left[ 3 \left( 2.1 \times 340 \right)^2 \right]^{1/2} \right\} \right)^{1/3}$$

$$= 0.0574 \text{ m} = 57.4 \text{ mm} \quad Ans.$$

With this diameter, we can refine our estimates for  $k_b$  and q.

Eq. (6-19): 
$$k_b = 1.51d^{-0.157} = 1.51(57.4)^{-0.157} = 0.80$$

Assuming a sharp fillet radius, from Table 7-1, r = 0.02d = 0.02 (57.4) = 1.15 mm.

Fig. (6-26): 
$$q = 0.72$$
  
Fig. (6-27):  $q_s = 0.77$ 

Iterating with these new estimates,

Eq. (6-32): 
$$K_f = 1 + 0.72 (2.7 - 1) = 2.2$$
  
 $K_{fs} = 1 + 0.77 (2.2 - 1) = 1.9$   
Eq. (6-18):  $S_e = 0.77 (0.80) (0.5) (560) = 172.5 \text{ MPa}$ 

Eq. (7-8): 
$$d = 57 \text{ mm}$$
 Ans.

Further iteration does not change the results.